Amendments to the Specification

Under the Title, Above Paragraph [0001], Replace the Section Heading as follows:

~~PRIOR ART~~ BACKGROUND OF THE INVENTION

Replace Paragraph [0001] with the following Amended Paragraph:

[0001] The invention starts with a gear drive unite, in particular for adjusting moveable parts in a motor vehicle~~, in accordance with the generic part of the main claim~~.

In Between Paragraphs [0002] and [0003], Replace the Section Heading as follows:

~~ADVANTAGES OF THE INVENTION~~ SUMMARY OF THE INVENTION

Replace Paragraph [0003] with the following Amended Paragraph:

[0003] The gear drive unit in accordance with the invention ~~with the features of the main claim~~ has the advantage that an axial force generating device is arranged in such a way that its coefficient of friction prevents a component that equalizes the axial play from receding radially. To do this, the geometry, the surfaces and the materials for the axial force generating device are selected in such a way that the coefficient of friction between a stopping face inclined by an angle of inclination against the perpendicular of the shaft and the surface of the component is greater than the tangent of the angle of inclination. In doing so, the component is displaced radially to the shaft as soon as the shaft has longitudinal play. Pushing back the component is prevented, however, by the frictional condition. As a result, an elastic element, which is used to displace the component, does not have to absorb any high restoring forces, which are initiated via the shaft on the component. Therefore, the elasticity of the elastic element is retained over its entire service life, thereby reliably eliminating the longitudinal play of the shaft over the entire service life.

Replace Paragraph [0005] with the following Amended Paragraph:

[0005] Advantageous developments of the device ~~in accordance with the main claim~~ are possible ~~due to the features specified in the sub-claims~~. Thus, the coefficient of friction between the surface of the component and the inclined stopping face is increased in an especially favorable way by forming a profile on one of the two friction surfaces. If, for example, a saw-tooth-like profile is formed on at least one of the surfaces, the component can be moved radially towards the shaft with less force, but can only be moved back radially again with a considerably higher expenditure of force. As a result, this type of structured surface leads to the elastic element for displacing the component not being excessively stressed. Therefore, the elastic element can be displaced back radially over the entire service life of the device in order to eliminate the axial play that is occurring. Because of forming such a profile on the friction surface between the component and the stopping face, the angle of inclination of the stopping face can be selected to be greater, thereby making greater travel available to equalize the shaft longitudinal ~~level~~play.
In a preferred embodiment, one of the two stopping faces or the component can feature a stair-step-like surface, in which the "stepping surfaces" are aligned to be approximately perpendicular to the longitudinal axis of the shaft. As a result, a restoring force of the component radially away from the shaft is practically completely prevented with the effect of a axial force from the shaft. This produces a situation where no shaft longitudinal play is permitted even in the case of extreme loads on the shaft.

Replace Paragraph [00017] with the following Amended Paragraph:

[00017] The angle of inclination of the stopping face can be enlarged by a saw-tooth profile, thereby making greater travel available to equalize the shaft longitudinal ~~level~~play.

In Between Paragraphs [00018] and [00019], Replace the Section Heading as follows:

~~DRAWINGS~~ BRIEF DESCRIPTION OF THE DRAWINGS

In Between Paragraphs [00030] and [00031], Replace the Section Heading as follows:

~~DESCRIPTION OF THE EXEMPLARY EMBODIMENTS~~ DETAILED DESCRIPTION

Replace Paragraph [00032] with the following Amended Paragraph:

[00032] The operating principle of this axial force generating device is depicted schematically in Figure 2. The stopping element 34 in this case is embodied as one piece with the component 44 so that the stopping face 35 is formed directly by the fore part 32 of the shaft ~~32~~18. When the shaft 18 is under load, an axial force 50 acts along the longitudinal axis 30 on the component 44, which passes on this axial force 50 to the stopping face 36. Resulting on the inclined stopping face 36 from the axial force 50 are a normal force 52 perpendicular to the stopping face 36 and a downhill slope force 54 parallel to the stopping face 36, which pushes back the wedge-shaped component 44 against the elastic element 48 from the gap 64 between the shaft 18 and the stopping face 36. A frictional force 56, which is generated when displacing the component 44 against the stopping face 36, acts against the downhill slope force 54. In order to prevent the axial force 50 from pushing the component 44 back against a displacement force 58 applied by the elastic element 48 in the case of a strong axial load of the shaft, according to the invention, the frictional force 56 is greater than the maximum occurring downhill slope force 54 in the case of maximum axial load of the shaft 18. This results mathematically in the tangent of the angle of inclination 40 being less than the coefficient of friction, which corresponds to the frictional force 56. The coefficient of friction in this case is essentially determined by the selection of material and the surface quality of the surfaces that can be displaced against each other.

Replace Paragraph [00035] with the following Amended Paragraph:

[00035] Figures 5a and 5b depict the component 44 from Figure 4 again in a side view and a top view. The friction surface 62 of the component 44 is arranged against the plane 42 by the same angle of inclination 40 as the corresponding stopping face 36 of the housing 15. The angle of inclination 40 and the overall length of the component 44 define a maximum travel 70 by which the shaft longitudinal play can be equalized at a maximum. Figure 5b depicts a maximum spring range 72 by which the elastic element 48 can be pre-stressed via the housing part 46 during assembly. This range ~~73~~72 results in the force 58 with which the elastic element 48 presses the component 44 into the gap 64.